

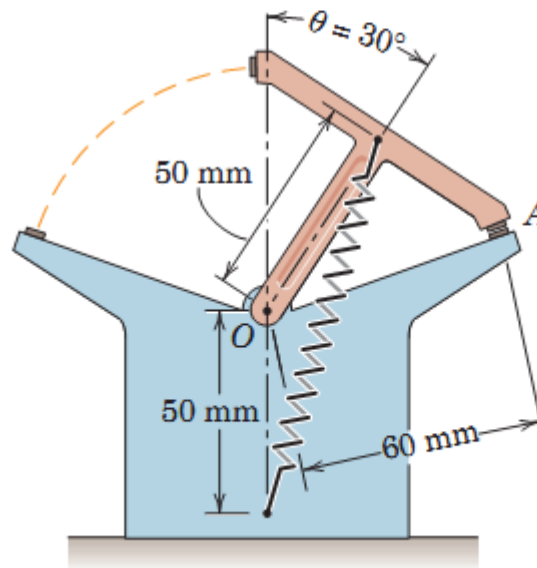
ME 206 – DYNAMICS – SPRING 2017

STUDY PROBLEMS-12

(PLANE KINETICS OF RIGID BODIES: WORK-ENERGY EQUATION)

**PROBLEM 6/121
(WORK-ENERGY)**

Specify the unstretched length l_0 of the spring of stiffness $k = 1400 \text{ N/m}$ which will result in a velocity of 0.25 m/s for the contact at A if the toggle is given a slight nudge from its null position at $\theta = 0$. The toggle has a mass of 1.5 kg and a radius of gyration about O of 55 mm . Motion occurs in the horizontal plane.



6/121

Let $l_0 =$ unstretched length of spring

$$\Delta V_e = \frac{1}{2}(1400)[(\overline{BC} - l_0)^2 - (0.10 - l_0)^2]$$

$$\overline{BC} = 2(0.05) \cos 15^\circ = 0.0966 \text{ m}$$

$$\Delta V_g = 0 \text{ (horizontal plane)}$$

$$v_A = 0.25 \text{ m/s}$$

$$\Delta T = \frac{1}{2} I_O \omega^2 = \frac{1}{2} (1.5) (0.055)^2 \left(\frac{0.25}{0.06} \right)^2 = 0.0394 \text{ J}$$

$$U' = 0$$

$$U' = \Delta T + \Delta V_e = 0$$

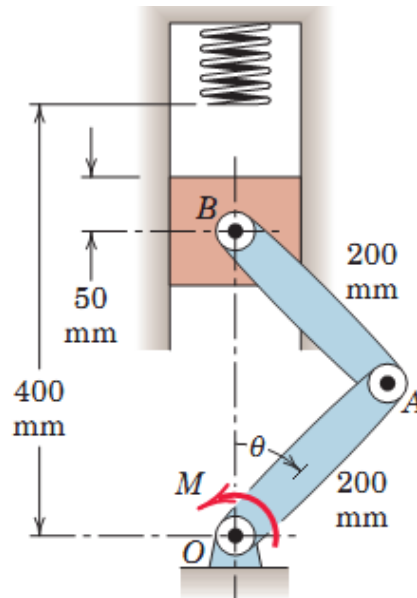
$$0.0394 + 700[(0.0966 - l_0)^2 - (0.10 - l_0)^2] = 0$$

$$0.0394 - 0.4690 + 4.77l_0 = 0, \quad l_0 = 0.0900 \text{ m}$$

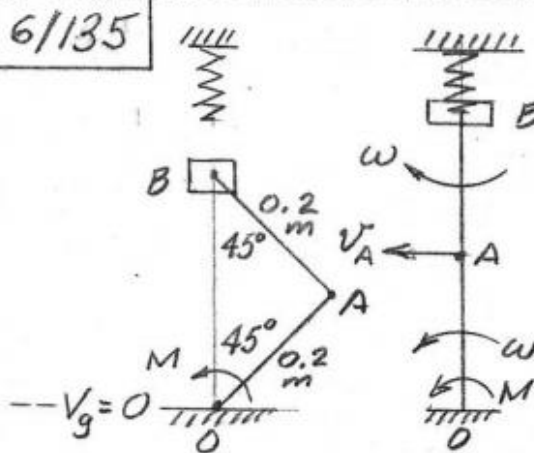
$$l_0 = \underline{90.0 \text{ mm}}$$

**PROBLEM 6/135
(WORK-ENERGY)**

Each of the two links has a mass of 2 kg and a centroidal radius of gyration of 60 mm. The slider at B has a mass of 3 kg and moves freely in the vertical guide. The spring has a stiffness of 6 kN/m. If a constant torque $M = 20 \text{ N}\cdot\text{m}$ is applied to link OA starting from the rest position at $\theta = 45^\circ$, determine the angular velocity ω of OA when $\theta = 0$.



6/135



$$\Delta V_e = \frac{1}{2} k x^2$$

$$= \frac{1}{2} 6000 (0.050)^2$$

$$= 7.5 \text{ J}$$

$$U = \int M d\theta = M\theta = 20 \frac{\pi}{4}$$

$$= 15.71 \text{ J}$$

$$U = \Delta T + \Delta V_g + \Delta V_e ; 15.71 = 2(0.0136 \omega^2) + (0.575 + 1.724 + 3.448) + 7.5$$

$$\omega^2 = 90.5, \omega = 9.51 \text{ rad/s}$$

Link OA;

$$\Delta V_g = 2(9.81)(0.1)(1 - \cos 45^\circ)$$

$$= 0.575 \text{ J}$$

$$I_0 = m(\bar{k}^2 + \bar{r}^2)$$

$$= 2([0.060]^2 + [0.1]^2)$$

$$= 0.0272 \text{ kg}\cdot\text{m}^2$$

$$\Delta T = \frac{1}{2} I_0 \omega^2 = 0.0136 \omega^2$$

Link AB;

$$\Delta V_g = 2(9.81)[0.3 - 0.3 \cos 45^\circ]$$

$$= 1.724 \text{ J}$$

$$\Delta T = 0.0136 \omega^2 \text{ same as OA}$$

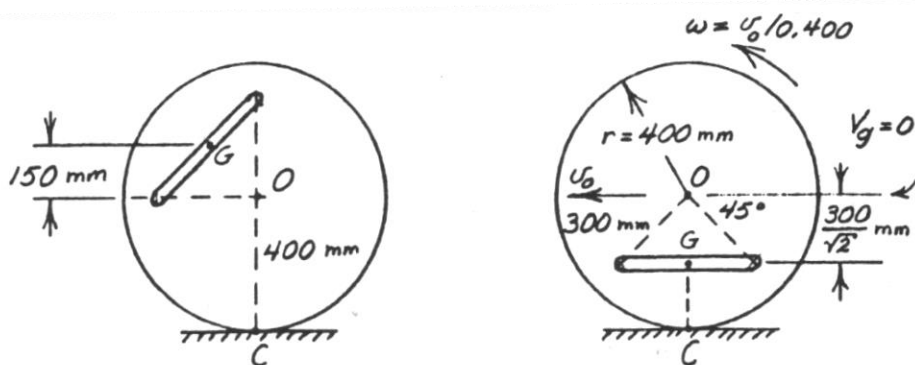
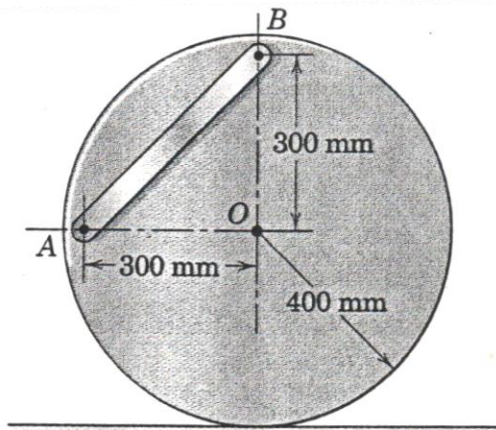
Block B;

$$\Delta V_g = 3(9.81)(0.4)(1 - \cos 45^\circ)$$

$$= 3.448 \text{ J}; \Delta T = 0$$

PROBLEM 6/153 (6th Ed.)
(WORK-ENERGY)

The 45-kg uniform circular disk with its attached 9-kg slender bar is released from rest in the position shown and rolls without slipping on the horizontal surface. Calculate the velocity v_O of the center O when the mass center of the bar is directly below the center O of the disk.



$$v_G = \frac{\overline{CG}}{r} v_O, \quad I_G = \frac{1}{12} 9(0.300\sqrt{2})^2 = 0.135 \text{ kg}\cdot\text{m}^2$$

$$U'_{1-2} = 0 = \Delta T + \Delta V_G \quad (1)$$

$$\begin{aligned} \Delta T_{\text{disk}} &= \frac{1}{2} 45 v_O^2 + \frac{1}{2} \left[\frac{1}{2} 45 (0.4)^2 \right] \left(\frac{v_O}{0.4} \right)^2 - 0 \\ &= 33.8 v_O^2 \end{aligned}$$

$$\begin{aligned} \Delta T_{\text{bar}} &= \frac{1}{2} m \bar{v}^2 + \frac{1}{2} \bar{I} \omega^2 = \frac{1}{2} 9 \left[\frac{0.4 - 0.3/\sqrt{2}}{0.4} v_O \right]^2 \\ &\quad + \frac{1}{2} 0.135 \left(\frac{v_O}{0.4} \right)^2 - 0 = 1.415 v_O^2 \end{aligned}$$

$$\Delta V_G = -mgh = -9(9.81)(0.150 + 0.3/\sqrt{2}) = -32.0 \text{ J}$$

$$\text{Eq. (1): } 0 = (33.8 + 1.415) v_O^2 - 32.0$$

$$\underline{v_O = 0.954 \text{ m/s}}$$